Vehicle Variable Estimation in Diagnostic Context

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Abstract. Safety in vehicles has many aspects and is implemented in different ways by manufacturers. With more safety systems to come, the vehicle will certainly start to have an operating system to manage the whole. Neural networks have an adaptative behavior that can be trained to meet new conditions and have a certain inherent degree of robustness when used as variable estimators. In this paper, we present a simplified model of the vehicle suitable to create neural network architectures that estimate the forces applied to the wheel as well as the vehicle body slip angle and yaw rate. For this purpose, we use the veDyna simulator which substitutes safely and economically real test vehicles. Typical extraneous and erroneous data are then presented to test the robustness of the network in order to judge the applicability of this approach from ideal, exact calculation conditions to real life situations.

1 Introduction

Safety is a topic that is being implemented more and more at many levels in vehicles thanks to advances in computational technologies. The key difference between these solutions is where they put the decision making, and along with it, the responsibility. An attempt in this regard is started with the IEEE P1616 "Motor Vehicle Event Data Recorders" draft standard. The first approach using systems such as Anti-lock Breaking System (ABS), Vehicle Stability Assist (VSA), Traction Control System (TCS) and Electronic Stability Program (ESP) [1], implement the control in the vehicle, leaving the driver passive in their function, except for his actions that put the vehicle in a condition or situation that activated these systems. The second approach [2] targets the behavior of the driver with respect to speed, tailgating and wearing the seatbelts. The infringement of a normal condition is reported by an audio-visual alarm of an increasing intensity as the vehicle speed exceeds the limit in the local area, the distance to the vehicle ahead is reduced and the seat belts is still off while the speed increases. The application of Neural Networks in a vehicle context is not new. Previous work has shown that the non linearity of a semi-active suspension can be overcome by the use of neural networks as control elements [3]. Also, the neural networks are more closely applied to parameter estimation through their use in measuring acceleration [4], and they are also used in a fault diagnostic context [5].

In this work, we propose to estimate the state of the vehicle based on a typical physical model, then we use MLP neural networks structures with one hidden layer to elaborate estimators of directly measurable or calculated variables. The justification

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of this approach is that with the proper training, this system can curve fit any real function [6], [7]. We then study the response of the estimation networks to data unseen before in the training. Robustness is probed by changing the driver and adding errors to the inputs.

2 Physical Model of the Vehicle

2.1 Vertical Reaction Force

Each wheel is subject to a vertical reaction force which is the resultant of the sum of all the small forces applied over the surfaces of contact between the tread and the road. This force is not necessarily applied at the centre of this surface and various wheel models try to locate and model it [8]. However, its magnitude can be determined by decoupling the front and rear axels and applying the laws of static and equilibrium of forces as shown on Fig. 1.

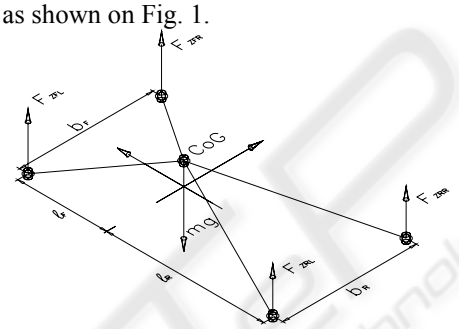

Fig. 1. Static Model of the Vehicle.

This way we can estimate the force on the rear left wheel F_{ZRL} to be:

$$F_{ZRL} = \frac{1}{2} m \left(\frac{l_F}{l} g + \frac{h_{CoG}}{l} a_X \right) - m \left(\frac{l_F}{l} g + \frac{h_{CoG}}{l} a_X \right) \frac{h_{CoG} a_Y}{b_R g} \tag{1}$$

where m is the mass of the vehicle; l_F, and l are the front and total lengths of the vehicle measured from the centre of gravity whose height is given by h_{CoG}; a_X, a_Y and g are the longitudinal, lateral and gravity accelerations; b_R is the length of the rear axle; *RL* refers to the Rear-Left wheel. In this equation, only a_X and a_Y are variables while the remaining elements are parameters. The height of the centre of gravity is constant for a given load only when the vehicle is standing still or moving at a constant velocity. However, it varies with the acceleration and thereby the accelerations remain the only variables needed for the estimation of this force.

2.2 Lateral Force

Due to the fact that the wheels on one side of the vehicle are not parallel to the other, and there is a small axis toe-in ($b_R \neq b_F$), a remnant lateral force F_Y is always present and it is equal and opposite for wheels on the same axel in straight course. This

embedded value will be trained into a neural network. The estimation will be limited to the cases when the vehicle is actually making a turn, represented by a steering angle larger than a given minimum. The horizontal and lateral forces are linked to the vertical reaction by the friction coefficient of adhesion:

$$\mu_x = \frac{F_x}{F_z} \text{ and } \mu_y = \frac{F_y}{F_z} \tag{2}$$

where F_X, F_Y and F_Z are the longitudinal, lateral and vertical forces applied to the wheel. Note that this ratio is constant only for the low values of the forces.

2.3 Wheel Slip

The relationship between wheel slip and the friction coefficient is the subject of various studies [9], [10]. Though the definition is the same in all references as being the difference between a wheel's rotational and translation speeds, as given by:

$$s = \frac{vdt - r\omega dt}{vdt} = \frac{v - r\omega}{v} = 1 - \frac{r\omega}{v} \tag{3}$$

where s is the slip, v is the vehicle's speed, r is the wheel radius, ω is the wheel's angular velocity and dt is an interval of time.

The forced rotation of the wheel generates the longitudinal force shown in Fig. 2.

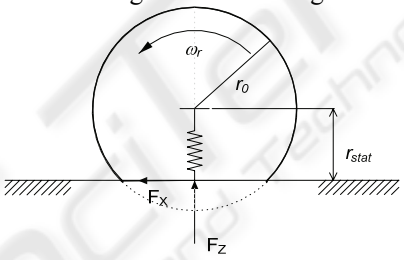

Fig. 2. Wheel's static radius and applied forces.

r_{stat} is the static wheel radius, defined as the distance between the centre of the wheel and the road for a given load at stand still. We note however that because each wheel is subject to a different vertical reaction force, they are compressed differently on the front and rear axles and therefore their radii are not the same. Any variation in this parameter implies a variation in the wheel's speed and we only consider the rotational speed as variable and the remaining elements as either parameters or implicit.

We conclude that the vehicle's speed, the wheel's rotational speed and the vertical force have the major influence in determining wheel slip.

2.4 Yaw Rate

We attribute to every wheel R_{ij} a rotational speed ω_{ij}. As the vehicle makes a turn about a central point, the angular movement around the vertical axis becomes proportional to the wheels' speeds, in either their rotational or linear form. Thus the yaw angle ψ is linked to the wheel rotation speed ω_{ij} and to the apparent turn radius of curvature conform indicated on Fig. 3.

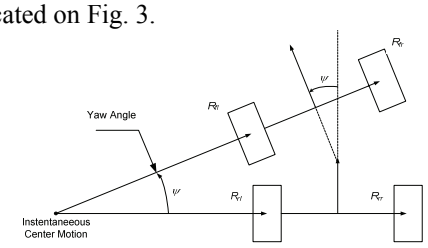

Fig. 3. Yaw movement in curve motion.

By observing this motion over time, we can establish the following relations:

$$\frac{d\psi}{dt} = \frac{r_{stat}\omega_{rl}}{R_{rl}} = \frac{r_{stat}\omega_{rr}}{R_{rr}} \tag{4}$$

where ψ is the angular yaw, r_{stat} is the wheel's static radius, ω_{rr} is the rotational speed of the rear right wheel, ω_{rl} is the rotational speed of the rear left wheel, R_{rr} is the radius of curvature of the path followed by the rear right wheel.

If need be, the yaw movement of the driveline can be upgraded to the vehicle's body where another component, an angle usually designated β measuring the side slip of this body as it tends to resist, or follow with a backlash, the velocity vector of the driveline. β is considered to be included in the yaw motion and without going through its relationship with the suspension system it can be estimated from the physical factors that cause it in the first place.

3 Simulated Driving Maneuvers

The German company TESIS Dynaware presents the vehicle simulation software veDyna [11]. This simulator takes the most recent mathematical models to calculate the behavior of the car step by step until the maneuver is over, thus providing accurate information on what may happen on the road under the given circumstances. The model we used is that of a BMW 325i model 1988, and we did not modify it in order to remain as close as possible to a real life situation. The veDyna simulation software uses about 1500 variables in order to render its virtual simulations close to real life conditions. With its Matlab interface and open connectivity though the declared functions, it offers unmatched flexibility allowing it to reproduce at a cheap cost a large variety of normal and abnormal driving situations. This way, it spares fastidious and dangerous maneuvers on real test vehicles. Typical maneuvers of slalom and acceleration-deceleration speed profiles were used to generate an extensive training

data set for neural network estimators. The performance of these networks is tested on a double lane change maneuver. In the following, we give a brief description of the training maneuvers, with emphasis on the cease acceleration maneuver used in testing the robustness against changing the driver.

3.1 Generation of the Training Database

In order to create the training and testing databases, we used the veDyna and Matlab software to generate several road profiles and maneuvers:

- A slalom profile is generated such that the course is sinusoidal while the driver maintains a constant velocity.
- A straight velocity profile is designed to solicit the vehicle in longitudinal accelerations and decelerations.

The data generated by the vehicle performing these maneuvers are gathered to form the training. In all, 56 variables were sampled at a period of 1ms, some of which have taken part in the training processes while others were used to draw and recreate the path followed by the vehicle on track. Thus we generated 5500 input vectors of 56 components each for the velocity profile maneuver and 4000 input vectors for the slalom maneuver. However, we did not use these sets in complete because we can eliminate the parts that correspond to a monotone or redundant situation on the road in order to accelerate the training process and reduce the amount of resources required by this task.

3.2 Generation of the Validation Database

The databases used to test the performance of the neural networks are generated using the Double Lane Change and Cease of Acceleration maneuvers described below. Theses maneuvers were chosen because they do not figure explicitly in the training database.

3.2.1 Description of the Double Lane Change Maneuver

The double lane change profile is an overtaking maneuver in which the driver changes lanes twice in order to overpass a virtual vehicle ahead. It is made of 2300 data samples.

3.2.2 Description of the Cease Maneuver

The Cease maneuver is mainly about releasing the acceleration pedal abruptly while in the middle of a curve. The driver starts at t = 0 on a straight course, and accelerates until the vehicle reaches the speed v_dep at time t_dep. This speed is maintained over a time t_hold until the driver goes on a constant acceleration phase for a period of t_acc. At this point, the vehicle is in the middle of a curve and the driver suddenly releases the acceleration pedal letting the vehicle roll for a time t_free before bringing it to a stop in an interval of time t_fin. The numerical value for the variables involved is given here for reference:

t_dep=10s t_hold=5s t_acc=5

t_free=5s t_fin=6s v_dep=72Km/h

The total maneuver time is 1s more than the sum of the parts because, as per the simulator's documentation the system must first start and remain idle until all calculation transients have elapsed [11]. A total of 3200 samples were generated for this maneuver. The driver model used to perform this maneuver is based on the fundamental primitives directly available from the simulator, mainly: (i) CRUISE and ACCPEDAL for longitudinal control and (ii) STRAIGHT, LINTABLE and FIXED for lateral control. In-depth information about these functions can be found in the referenced documentation. The main difference between these directives and the "advanced driver" model available in veDyna is that while the former behaves simply as it is told, the latter uses PID control to maintain the centre of gravity of the vehicle aligned with the centre line of the road path, while observing given temporal or special constraints.

4 Experimental Results

In this paragraph we use the Multi-Layer Perceptron architectures with one hidden layer to estimate the vertical reaction force F_Z, lateral force F_Y, wheel slip and the vehicle yaw. For our application, we apply the BFGS training algorithm described in [12]. The results are compared to the values generated by the simulator, hence substituting safely and more economically a real vehicle. Note that it is a good strategy to test the ability of the network to estimate the desired function by hiding known, simulator generated, data in the training phase and leave it for guessing at a later stage. One more reason for doing so is that the data in our test sets are within the extreme limits already fed to the network through the training sets; for example, the turns in slalom are harder and the speed and breaking in the velocity profile are more abrupt than the ones in the double lane change, for example. Finally, besides straight driving, overtaking or double lane changes is one of the most performed maneuvers on the road.

The cease acceleration maneuver is also very common and hence it is used in testing the robustness of the neural networks against a change in the driver model.

4.1 Test of Vertical Reaction Force F_Z

Each wheel has a dedicated neural network made of 2 inputs a_X and a_Y, 19 neurons in the hidden layer and one output.

The results comparing the output of this network to the output of the simulation software in the case of the double lane change maneuver for the rear left wheel are given in Fig. 4. In this case, the estimator is able to calculate Fz accurately when the vehicle body is not strongly sollicitated, otherwise it will commit a maximum error of 10%. Note that all simulation figures, and in this case Fig. 4, are a snap shot of continuous motion over the whole simulation time, which can sometimes go around one minute of driving and contains too much of non critical information to show on a figure. On the other hand, we are observing 56 variables at veDyna's default calculation step of 1ms, so the software anticipated that this large size of 10 bytes

IEEE Real variables is not entirely necessary to store in a log file, so we use its default rate of 1 in every 10 samples for observing the variables. Hence, the 0 in x-axis of the figures corresponds to the time t0 as of which the data starts to be interesting for us and the sampling time is to be multiplied by 10 to read the real simulation time; thus 200 corresponds to 2000ms after t0.

4.2 Test of the Lateral Force F_Y

The inputs are the lateral acceleration a_Y and the steering angle lrw, with 40 neurons in the hidden layer and one linear output. The results of the estimation of this force on the double lane change maneuver are shown in Fig. 5, with an average error of 15.8%.

4.3 Test of Wheel Slip "s"

Each wheel has a dedicated neural network of its own, having 3 inputs and 53 neurons in the hidden layer. The longitudinal component of the speed of the vehicle's centre of gravity is a common input to all four networks; the remaining two inputs are the specific wheel's rotational speed and the applied vertical force $F_Z \equiv F_{ZXY}$, with XY=RR, RL, FL or FR. The results of the simulation on the double lane change profile are shown in Fig 6. We conclude that the wheel slip can indeed be calculated with a ±6% tolerance from the vehicle's overall speed and the wheel's rotation speed under given load conditions F_{ZXY}.

It can be inferred from the Fig. 6 that the maximum slip is well below 10% and we are in the linear edge of the slip curves, so a less complicated approach is feasible. Yet it should be noted that the value of sl is only small because veDyna's "advanced driver" is programmed to keep the vehicle near its optimal functional point, where it has the most traction (and hence stability) on the road and the efficiency of the engine as seen from the wheels is maximal. A human driver will not always be able to do so and will need to be followed throughout the entire slip margin.

4.4 Test of Yaw Rate "dψ/dt"

The inputs to the neural network are three of the four wheels' rotational speed, since the fourth wheel is forcibly dragged by these three and the vehicle body since it is not deformable. However, we noted that the network is better run if it is given the difference between these variables, two by two, instead of giving them in raw format and let the network figure it out. This idea was used because in order to obtain a rotation, the wheels' speed must be all different, and the bigger this difference is the more the yaw. This way we formed the difference: ω_{FL}-ω_{FR} and ω_{FL}-ω_{RL}. The third input is the lateral acceleration. The hidden layer contains 37 neurons and the output is shown in Fig. 7. We observed that the estimation of the yaw rate is within a good ±5% tolerance, mainly due to the influence of geometry on this value, rather than internal variables and given that the slalom training set is exhaustive in this respect with regard to the large yaw rate values it contains.

4.5 Simulation Results – Figures

Following are the figures summarizing the results discusses above, where the discussion given in §4.1 applies.

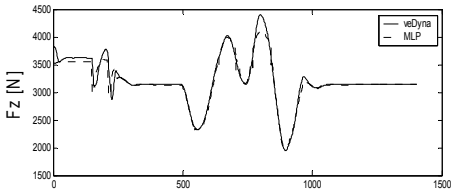

Fig. 4. Estimation of the vertical reaction force.

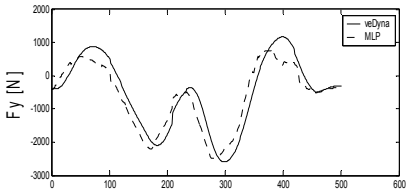

Fig. 5. Estimation results for F_Y.

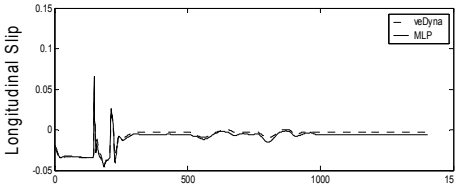

Fig. 6. Estimation of wheel slip s.

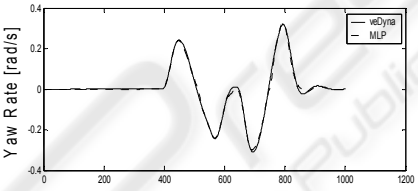

Fig. 7. Estimation of Yaw Rate dψ/dt.

4.6 Summary

The neural networks described above share a lot of common inputs; therefore they can be modularly integrated in a single structure as shown in Fig. 8.

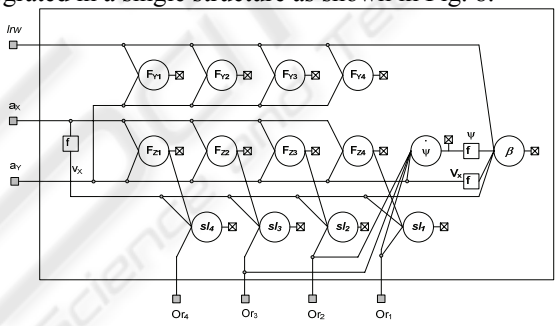

Fig. 8. Integrated Neural Network for Vehicle Variable Estimation.

For example, all four networks that calculate the lateral force F_Y on the wheel have the same inputs: the steering wheel angle and the lateral acceleration, the only difference between them is the weights; due to the way each one was trained. Thus we need to read the value of lrw and a_Y only once and from there we calculate 4 outputs. The same is generalized for the other networks described in this paper.

5 Robustness of MLP Networks

5.1 Test Against Change in the Driver Model

Neural Networks have been used to improve driver models [12], [13]; however we limit our study to the PID driver model implemented in veDyna.

This test is done using the Cease maneuver presented above. The a_X and a_Y components of the acceleration are passed to the same estimator of the vertical force F_Z. The result is shown on Fig 9.

As we can see, the network is able to follow the target result with a total error of 3.86%, but inconsistency appears around the point where the driver suddenly releases the acceleration pedal, because the network was not trained for this event.

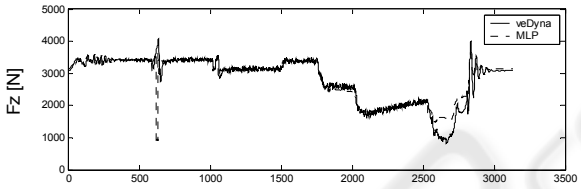

Fig. 9. Estimation results of F_Z on the Cease maneuver.

5.2 Test Against Errors at the Inputs

The robustness the MLP networks against errors at their inputs is studied using the MLP estimator of Fz applied to the double lane change maneuver as a typical example. The source of error on the inputs can be due to either background noise, inaccurate sensors or to the method of measurement itself.

We will take the acceleration components a_X and a_Y from the double lane change maneuver and add as much as 20% of uncertainty to the measured values of a_X and a_Y, as per the following equation:

```
a=(0.8+0.4*rand(2, data_size)).*a;
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(5)

where a is the acceleration vector. The effect of this modification on the longitudinal acceleration is shown on Fig 10 and the estimated force Fz comes with an overall error of 1.9% as shown on Fig. 11.

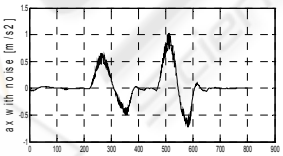

Fig. 10. Effect of 20% inaccuracy on a_Y.

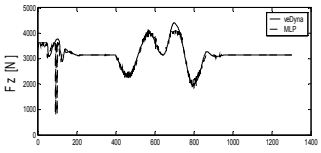

Fig. 11. Estimation of F_Z with input error.

Similar tests were repeated on other networks with results equivalent in nature, however an exhaustive study of the tolerance limit is put for future work.

6 Conclusion

We used simulated driving maneuvers to test MLP networks for parameter estimation in a vehicle supervision context. For this purpose, the vertical reaction force, the yaw rate and the wheel slip have been estimated by neural network systems. The choice of inputs for the MLPs was inspired from the physical model and the size of the hidden layers was fixed after an exhaustive range scan up to 60 neurons. With regard to robustness, and although only a limited number and somewhat particular cases were studied, all results show that neural networks have an inherent degree of immunity towards various types of errors. Thus this work concludes that a vehicle operating system can make use of MLP estimation networks as inputs. This system can be complementary to the multimedia and GPS services now being offered to passengers. Its implementation should benefit from the speed of widely available neural network integrated circuits, leaving any bottleneck to the upper software level.

Finally, the estimation results are best when the data under test is generated by the same driver as the one used in the training phase. Therefore it is essential that the networks be trained, under generic procedures, by the drivers that will use them.

References

1. Robert Bosch GmbH: ESP Electronic Stability Program, attributed to group effort (1999)
2. Sala, B.: Digital Device Development Group, Implementing Intelligence in Vehicles, The Magazine of Engineers Australia, Vol. 76, N° 9, (2004) pp. 26-29
3. Guo, D. L., et al.: Neural Network Control for a Semi-Active Vehicle Suspension with a Magnetorheological Damper, Journal of Vibration and Control, Vol. 10, No. 3, (2004) pp. 461-471
4. Gao, X. Z., et al: Acceleration signal estimation using neural networks, Measurement. Science and Technology 12, (2001) pp. 1611-1619
5. He, Y., et al.: Engine Real-time Fault Diagnosis using Neural Networks, Automation Technology for Off-Road Equipment, Proceedings of the July 26-27 Conference, Chicago, Illinois, USA (2002), pp. 89-95
6. Weigl, K., Berthod, M.: Neural Networks as Dynamical Bases in Function Space, INRIA, report N° 2124 (1993)
7. Hornik, K., et al.: Multilayer Feedforward Networks are Universal Function Approximators, IEEE Transactions on Neural Networks, Vol 2, No. 5, (1989) pp. 359-366
8. Kiencke and Nielsen: Automotive Control Systems, SAE International, Springer (2002)
9. Canudas, C.: Dynamic Tire Friction Models for Vehicle Traction Control, Laboratoire d'Automatique de Grenoble, Conference on Decision and Control, Phoenix, Arizona (1999)
10. Kim, Y. S., et al.: Development of RT vehicle simulation system for integration of ABS HIL and a driving Simulator, Department of Automotive Engineering, Kookmin University, Seoul, Korea.(1998)
11. TESIS DYNAware veDyna Example Book - Standard Road (2002)
12. Rivals, I., et al.: Real-time control of an autonomous vehicle: A Neural Network Approach to the Path Following Problem, 5th International Conference on Neural Networks and their Applications, NeuroNîmes (1993)
13. Lin, Y., et al.: Artificial neural network modeling of driver handling behavior in a driver-vehicle-environment system, International Journal of Vehicle Design - Vol. 37, No.1, (2005) pp. 24 – 45